



# AUTOMATIC DOG FEEDER

# HELLO!

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# THE PROBLEM

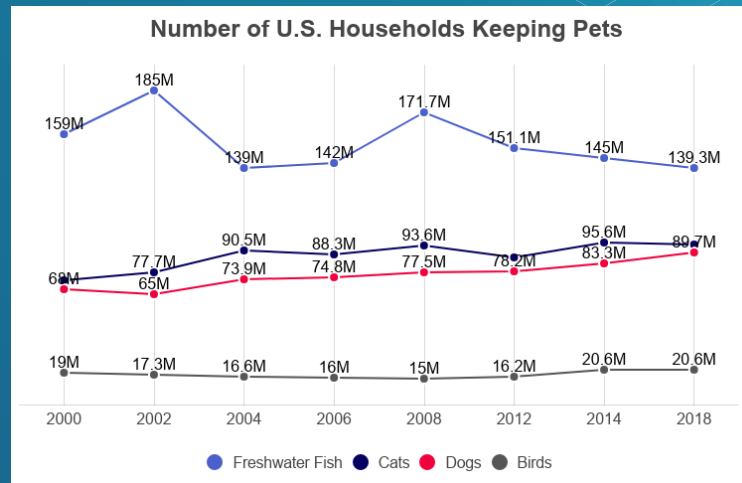
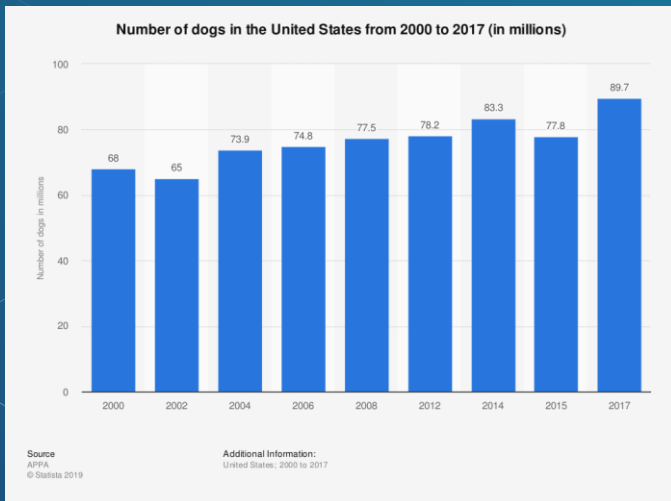
INTRODUCTION

A white double quote icon inside a dark blue hexagonal shape, which is part of a vertical blue gradient bar at the top center of the slide.

According to statistics, the number of dog owners is increasing over and over

# STATISTICS

- ◆ Number of dogs in the U.S. 2000 - 2017
- ◆ Number of dogs compared to other pets

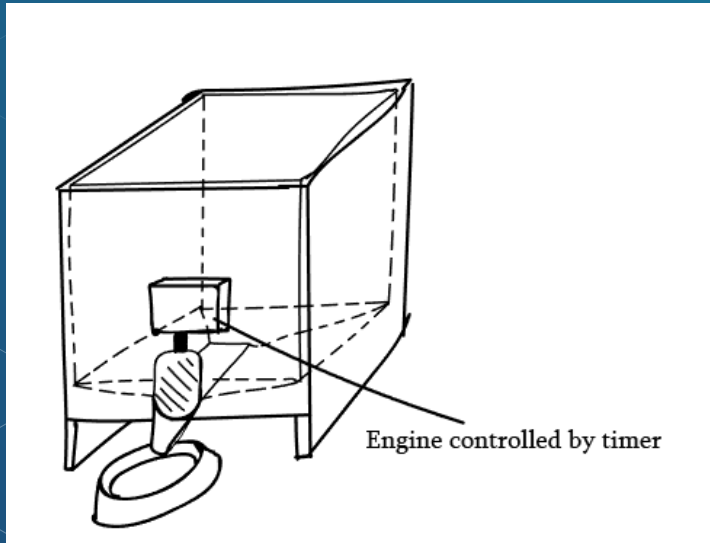


# DESIGN

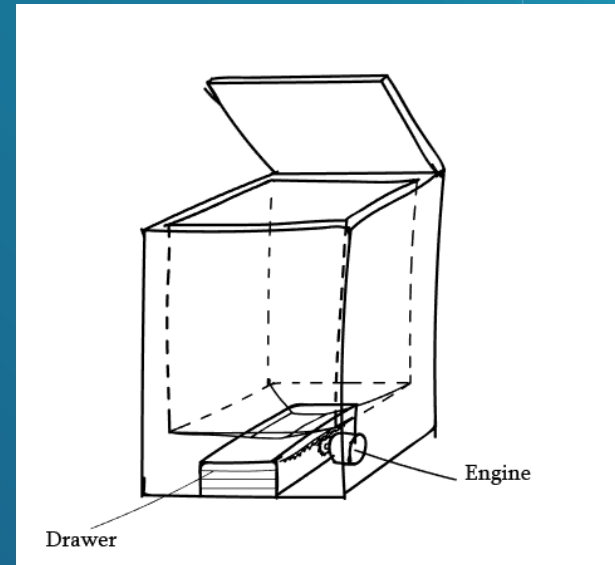


# DESIGN OPTIONS

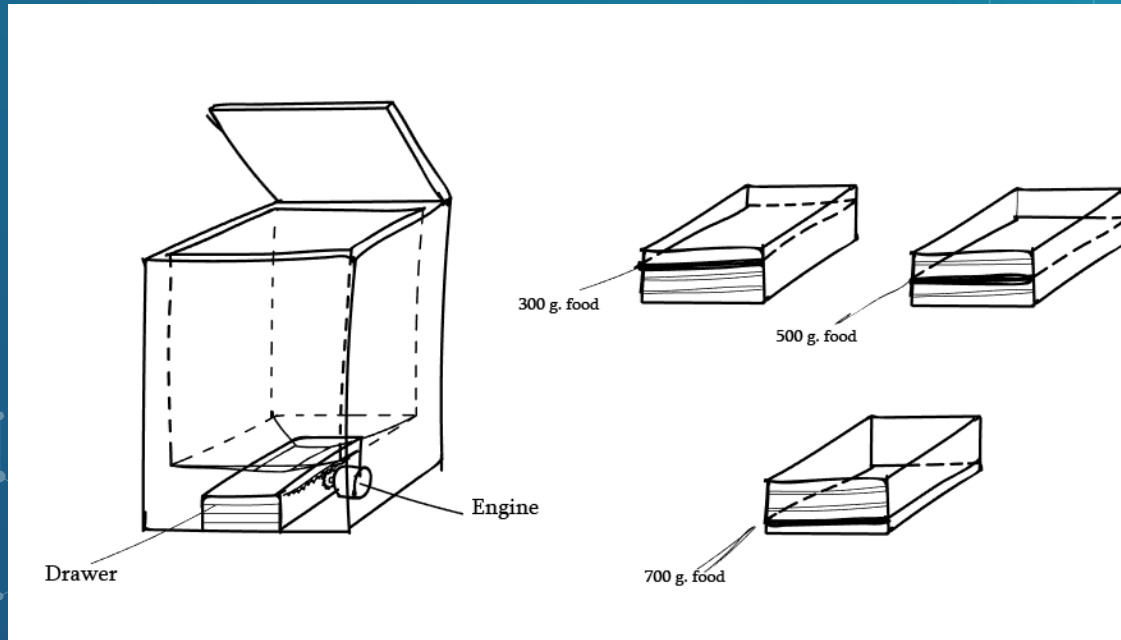
Little door which is controlled by electric timer



The second one is equipped with its own drawer



# CHOOSEN DESIGN





# MECHANISMS

- ◆ RACK AND PINION
- ◆ CAM AND FOLLOWER



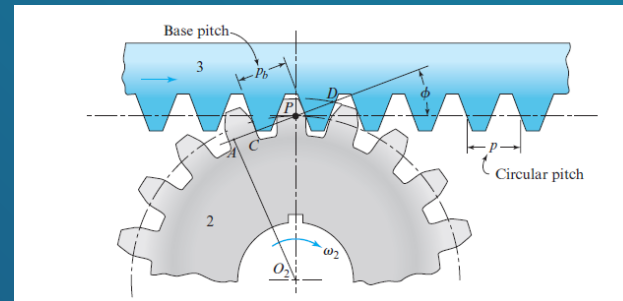
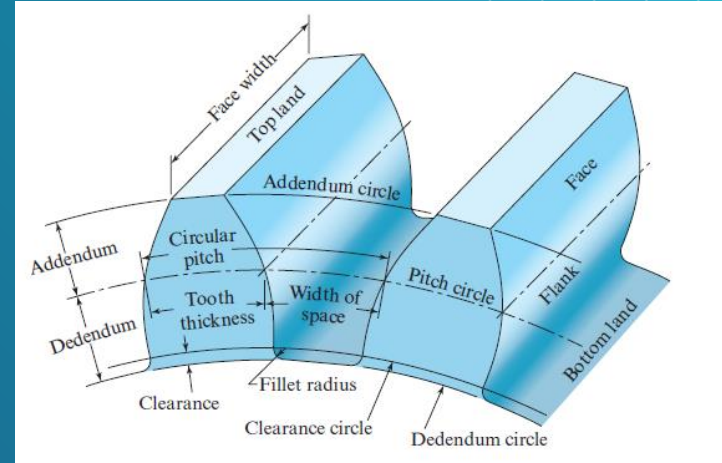
# RACK AND PINION

- ◆ They are characterized with perfect durability and are very compact in size



# PARAMETERS

- ◆ Module -  $m = P_t/\pi$  (Input Value)
- ◆ Number of teeth -  $Z_1/Z_2$  (Input Value)
- ◆ Pitch circle diameter -  $d_w = mz$
- ◆ Pitch angle -  $\alpha$
- ◆ Height of teeth -  $h_d = 2m$
- ◆ Fillet radius -  $r_i = 0.4m$
- ◆ Addendum height -  $h_{wa} = m$
- ◆ Dedendum height -  $h_{wf} = m$
- ◆ Pitch -  $P_t = m\pi$
- ◆ Addendum circle -  $d_a = m(z+2)$
- ◆ Dedendum circle -  $d_f = m(z-2.5)$



# OUTPUT PARAMETERS

CALCULATED WITH THE SOFTWARE CODE WRITTEN BY ME IN C++

- ◆ Height of tooth -  $h_d = 2m = 4\text{mm}$
- ◆ Fillet radius -  $r_f = 0.4m = 0.8\text{mm}$
- ◆ Addendum height -  $h_{wa} = m = 2\text{mm}$
- ◆ Dedendum height -  $h_{wf} = m = 2\text{mm}$
- ◆ Pitch -  $P_t = m\pi = 6.28\text{mm}$
- ◆ Addendum circle -  $d_a = m(z+2) = 38\text{mm}$
- ◆ Dedendum circle -  $d_f = m(z-2.5) = 29\text{mm}$



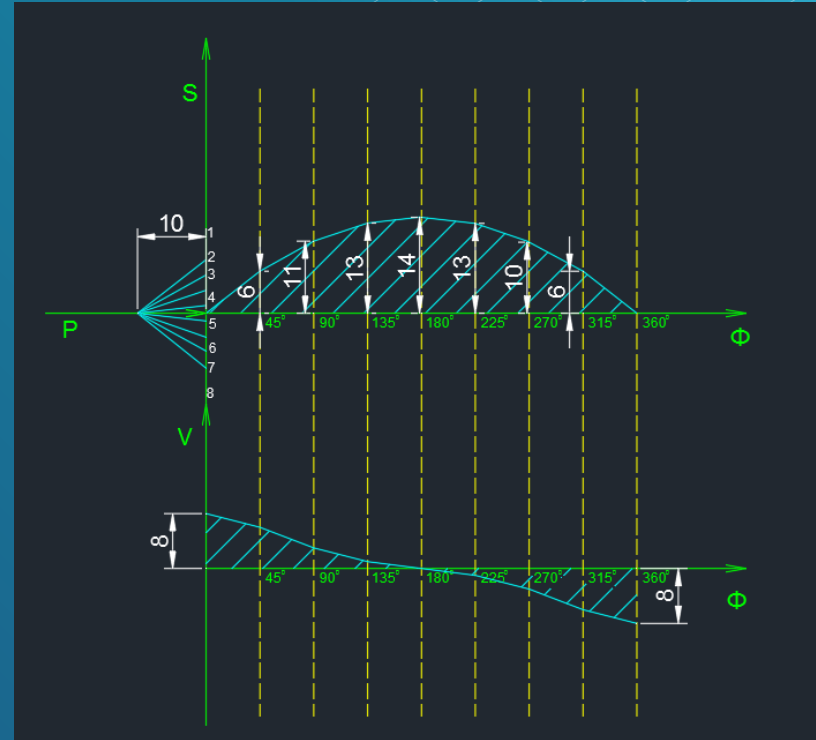
# CAM AND FOLLOWER

- ◆ The universality and flexibility in the design are among their most attractive features



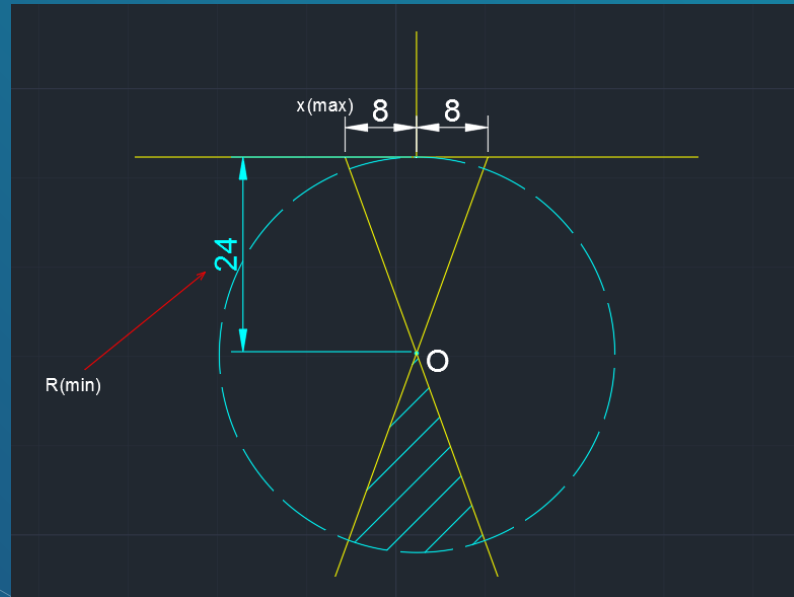
# GRAPHICAL DERIVATION

- ◆ Equation of Follower motion.  
Derived from Lagrange interpolation polynomial :
- ◆  $L(x) = -14 * (x^2 - 6.28x)/9.85$



# OPTIMAL RADIUS

- ◆ Every point under hatched plane can be used for Cam center point

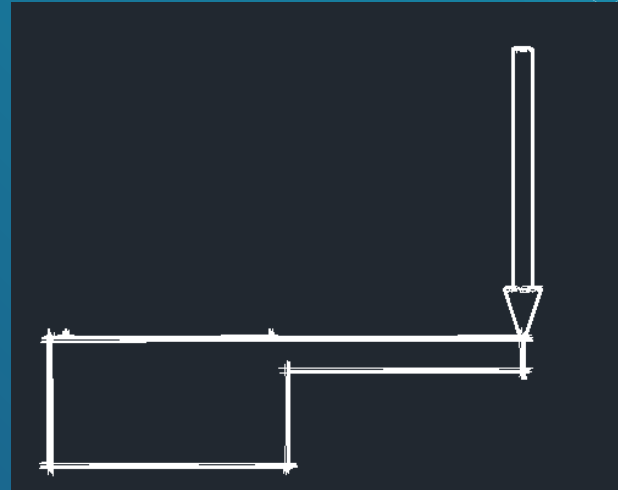
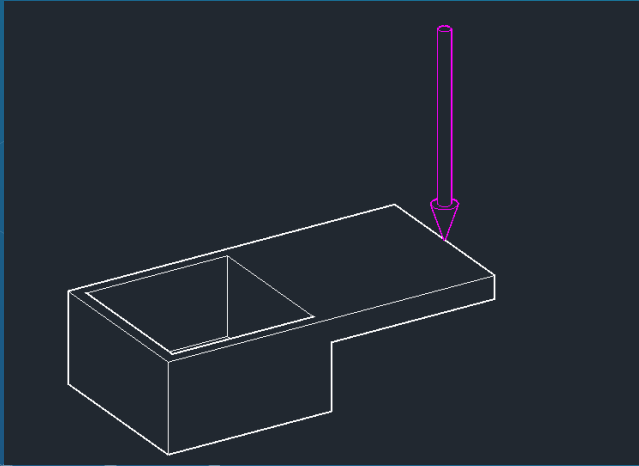


# RACK AND PINION vs. CAM AND FOLLOWER

	R.&P.	C.&F.
MANUFACTURING COSTS		
COMPACT IN SIZE		
POWER USAGE		

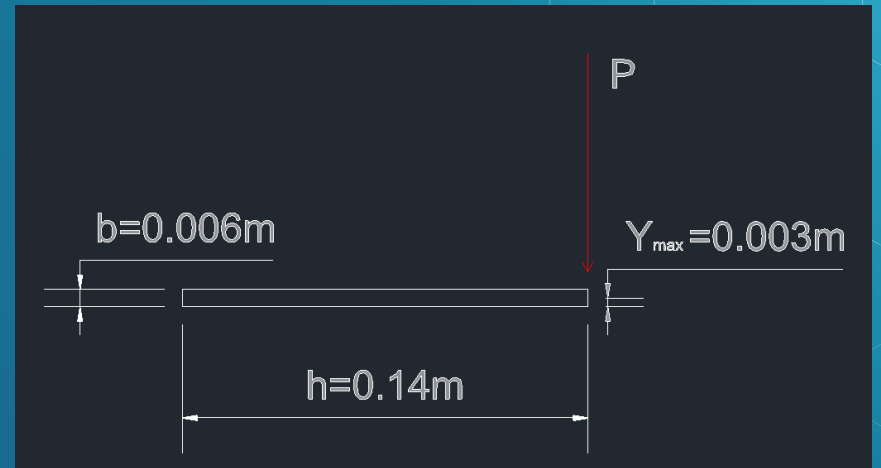


# MAX. POSSIBLE STRAIN

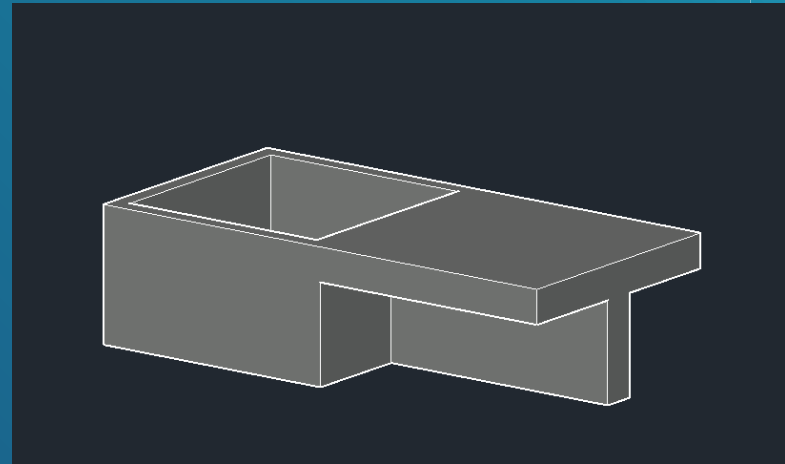
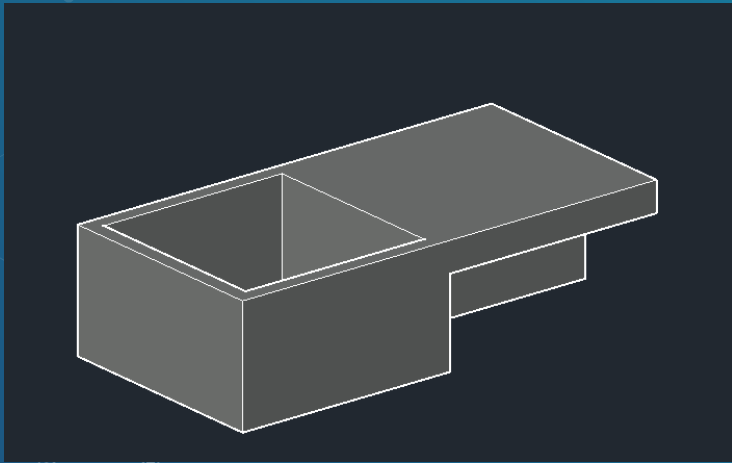


# STRAIN CALCULATION

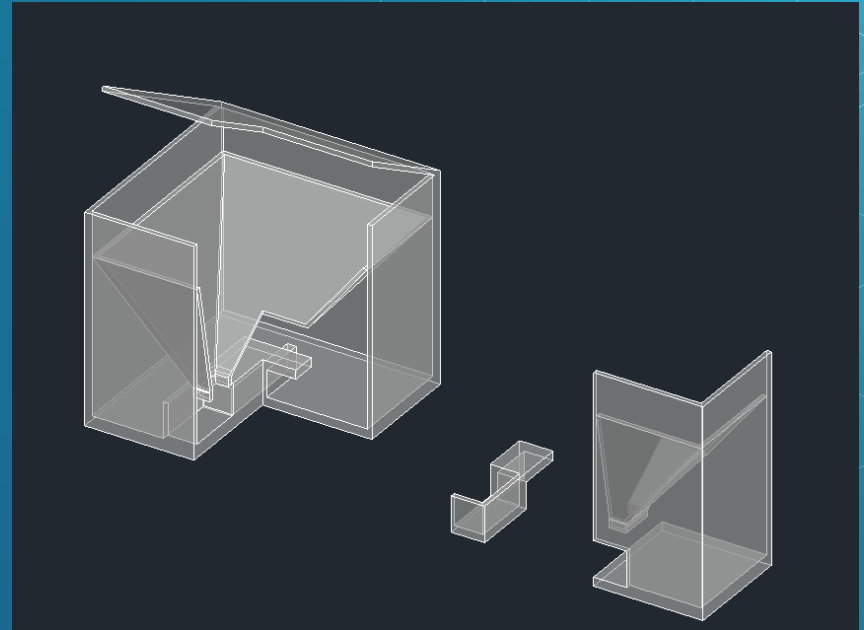
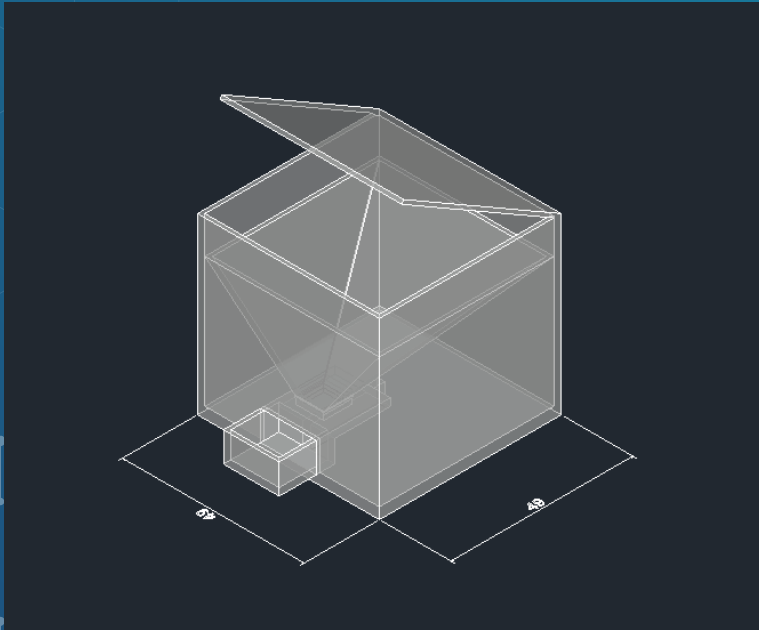
- Maximum possible strain  $[\delta] = 10/196 = 0.051 \text{ (kg/cm}^2\text{)} = 5003 \text{ (N/m}^2\text{)}$
- $I_x = b \cdot h^3 / 12 = 137.2 \text{ (cm}^4\text{)} = 1.372 \times 10^{-6} \text{ (m}^4\text{)}$
- $\delta_{\max} = M_x \cdot y_{\max} / I_x = 140 \cdot 0.3 / 137.2 = 0.3 \text{ (kg/cm}^2\text{)} = 29400 \text{ (N/m}^2\text{)}$



# MINIMIZING MAXIMUM STRAIN



# AUTOCAD PROTOTYPE



# CONCLUSION

## TASKS DONE:

- OPTIMAL DESIGN
- CHOOSING AND CALCULATING OPTIMAL MECHANISM
- STRAIN CALCULATIONS

## TASKS IN DEVELOPMENT

- ENGINE SELECTION
- ELECTRICAL PART
- PROTOTYPE CREATION

# THANKS!

ANY QUESTIONS?

